

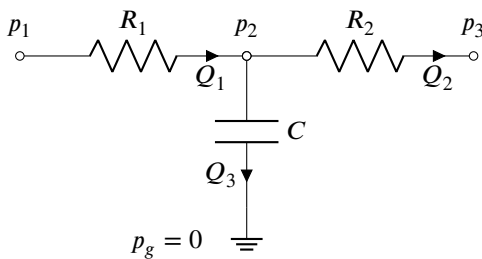
7.1 Lumped Parameter Modelling

Resistance, Compliance, and Inertance

Resistance	Compliance	Inertance
$Q = \Delta p/R$	$Q = C \frac{\partial p}{\partial t}$	$p = L \frac{\partial Q}{\partial t}$

- **Resistance R** : analogous to the electrical resistance which models the dissipation of energy. The mass flow rate Q is analogous to the electrical current (usually denoted by I), and the pressure p is analogous to the electrical voltage (usually denoted by V).
- **Compliance C** : this models the expansion of cardiovascular chambers under pressure, allowing it to store more fluid.
- **Inductor L** : this models the inertance of the fluid. When the fluid momentum is substantial, as the pressure on forward-flowing fluid reverses, the fluid will not suddenly reverse its direction, but decelerate over a transient.

Solving a Lumped Parameter Network Consider the example lumped parameter network,



... which yields a linear system with 4 unknowns (p_2, Q_1, Q_2, Q_3) and 4 simultaneous equations:

$$\begin{cases} p_2 - p_1 = R_1 Q_1, \\ p_3 - p_2 = R_2 Q_2, \\ Q_3 = C(p_2^{(t)} - p_2^{(t-1)})/\Delta t, \\ Q_1 = Q_2 + Q_3. \end{cases}$$

Note that $p_2^{(t-1)}$ denotes the pressure p_2 at the previous time step $t - 1$; $(p_2^{(t)} - p_2^{(t-1)})/\Delta t$ is an expression of the time derivative in the backward Euler fashion. (cf. electrical capacitor $I = C \cdot dV/dt$).

The above linear system can be arranged into a matrix system, $\mathbf{Ax} = \mathbf{b}$,

$$\begin{bmatrix} 1 & -R_1 & 0 & 0 \\ -1 & 0 & -R_2 & 0 \\ -1 & 0 & 0 & \frac{\Delta t}{C} \\ 0 & -1 & -1 & -1 \end{bmatrix} \begin{bmatrix} p_2 \\ Q_1 \\ Q_2 \\ Q_3 \end{bmatrix} = \begin{bmatrix} p_1 \\ -p_3 \\ -p_2^{(t-1)} \\ 0 \end{bmatrix},$$

and can be easily solved by inversion of the coefficient matrix: $\mathbf{x} = \mathbf{A}^{-1}\mathbf{b}$.

7.2 Windkessel Models

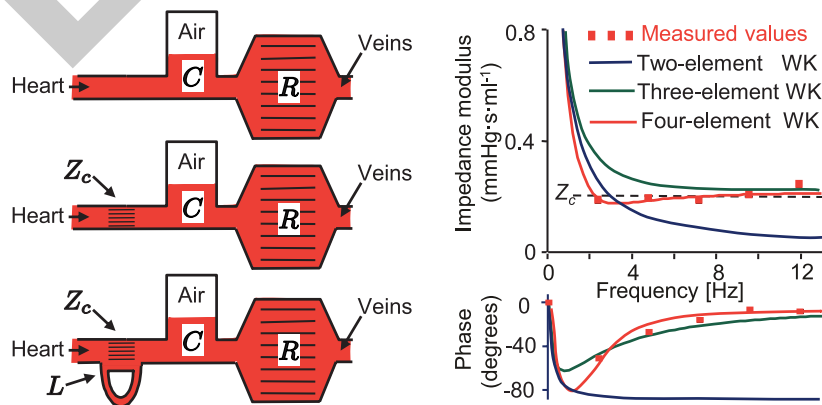
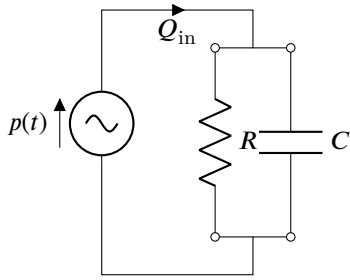


FIG. 1: Left: the mechanical equivalence of three Windkessel models; Right: Input impedances of the three Windkessels compared with the measured input impedance. (Westerhof *et. al*)

2-element Windkessel Model

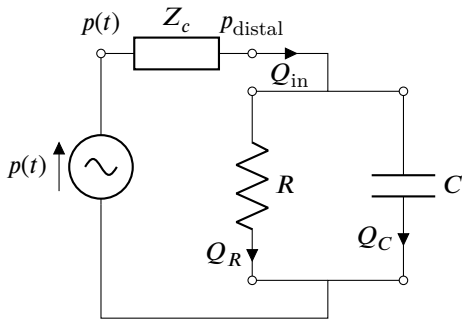


Governing Equation:

$$\frac{dp(t)}{dt} + \frac{p(t)}{RC} = \frac{Q_{in}}{C}$$

where C denotes the vessel compliance (elasticity), R denotes the peripheral (distal) resistance.

3-element Windkessel Model



Governing Equation:

$$\frac{\partial p(t)}{\partial t} + \frac{p(t)}{RC} = \frac{Q_{in}}{C} \left(1 + \frac{Z_c}{R}\right) + Z_c \frac{\partial Q_{in}}{\partial t}$$

where Z_c is the characteristic impedance, $p(t) - p_{distal} = Z_c Q_{in}$.

Derivation

Apply Kirchhoff's Current Law at node p_{distal} : $Q_{in} = Q_R + Q_C$. Moreover, since $p(t) - p_{distal} = Z_c Q_{in} \Rightarrow p_{distal} = p(t) - Z_c Q_{in}$.

- Current passes through the resistor R :

$$Q_R = \frac{p_{distal}}{R} = \frac{p(t) - Z_c Q_{in}}{R} = \frac{p(t)}{R} - \frac{Z_c Q_{in}}{R}$$

- Current passes through the capacitor C :

$$Q_C = C \frac{\partial p_{distal}}{\partial t} = C \frac{\partial [p(t) - Z_c Q_{in}]}{\partial t} = C \frac{\partial p(t)}{\partial t} - CZ_c \frac{\partial Q_{in}}{\partial t}$$

Hence, the total flow Q_{in} is

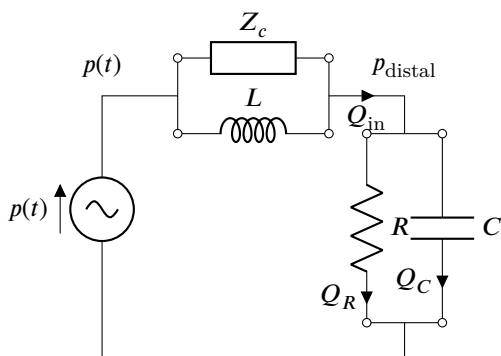
$$\begin{aligned} Q_{in} &= Q_R + Q_C \\ &= \frac{p(t)}{R} - \frac{Z_c Q_{in}}{R} + C \frac{\partial p(t)}{\partial t} - CZ_c \frac{\partial Q_{in}}{\partial t}, \end{aligned}$$

rearrange, we get

$$C \frac{\partial p(t)}{\partial t} + \frac{p(t)}{R} = \left(1 + \frac{Z_c}{R}\right) Q_{in} + CZ_c \frac{\partial Q_{in}}{\partial t}$$

Divide both sides of the equation above by C , we will get the final governing equation as presented.

4-element Windkessel Model



Governing Equation:

$$\frac{\partial p}{\partial t} + \frac{p(t)}{RC} = \frac{Q}{C} \left(1 + \frac{Z_{total}}{R}\right) + Z_{total} \frac{\partial Q}{\partial t}$$

where $Z_{total} = \frac{j\omega L Z_c}{j\omega L + Z_c}$ is the total impedance of the parallel network - the characteristic impedance, Z_c and the inductor, L .

Derivation

Apply Kirchhoff's Current Law at node p_{distal} : $Q_{\text{in}} = Q_R + Q_C$. However, we need to express p_{distal} in terms of $p(t)$, hence need to solve the total impedance of the Z_c - L parallel network:

$$\frac{1}{Z_{\text{total}}} = \frac{1}{Z_c} + \frac{1}{j2\pi f L} = \frac{j2\pi f L + Z_c}{j2\pi f L Z_c} \implies Z_{\text{total}} = \frac{j2\pi f L Z_c}{j2\pi f L + Z_c}$$

Note that sometimes $2\pi f$ is denoted as ω , which is the angular frequency. Now, $p(t) - p_{\text{distal}} = Z_{\text{total}} Q_{\text{in}}$. The rest of this derivation follows the same procedure for 3-WK.

Necessity of the inductance in 4-WK? Better capture the frequency characteristics of the flow.

- At the low f range: $2\pi f L \ll Z_c$, hence $Z_{\text{total}} \rightarrow 0$, which removes the characteristic impedance in the whole circuit;
- At the high f range: $2\pi f L \gg Z_c$, hence $Z_{\text{total}} \rightarrow Z_c$.

This means the inductance has no effect when the flow is steady, providing a zero resistance pathway to the rest of the circuit under steady flow conditions.

7.3 Moens-Korteweg Model of Pulse Wave Velocity

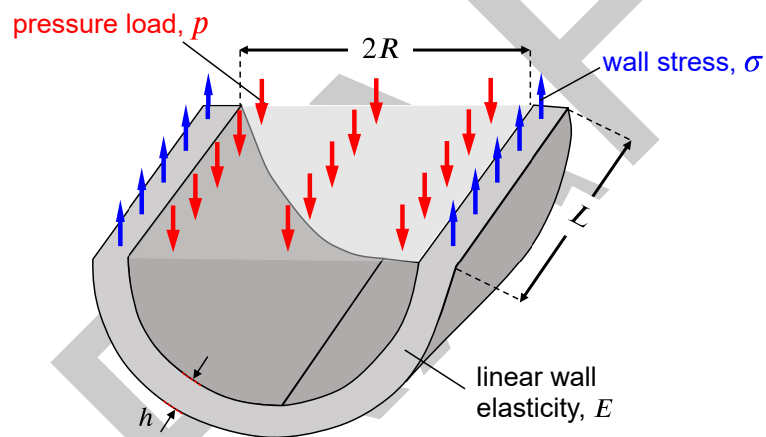


FIG. 2: The schematic for the derivation of Moens-Korteweg equation.

Equation 1 Assume linear elasticity (fixed Young's modulus, E), the stress(σ)-strain(ϵ) relation is

$$\sigma = E\epsilon = E \frac{\Delta R}{R} \quad \text{with} \quad \epsilon = \frac{(2\pi(R + \Delta R)) - 2\pi R}{2\pi R} = \frac{\Delta R}{R}$$

Applying Newton's 2nd Law and re-arranging the expression leads to an expression of the pressure,

$$m_{\text{wall}} a_{\text{wall}} = F_{\text{pressure}} - F_{\text{wall}}$$

$$0 = 2RL \times P - 2Lh \times \sigma \implies P = \frac{\sigma h}{R} = \frac{Eh}{R^2} \Delta R$$

Differentiating p w.r.t. t , this leads to **equation 1**,

$$\frac{\partial p}{\partial t} = \frac{Eh}{R^2} \frac{\partial \Delta R}{\partial t}$$

Equation 2 Integrating the continuity equation over the vascular cross-sectional area

$$\begin{aligned}
 & \text{0, axis-symmetrical} \\
 & \frac{1}{r} \frac{\partial ru_r}{\partial r} + \frac{1}{r} \frac{\partial u_\theta}{\partial \theta} + \frac{\partial u_z}{\partial z} = 0 \Rightarrow \int \left(\frac{1}{r} \frac{\partial ru_r}{\partial r} + \frac{\partial u_z}{\partial z} \right) dA = 0 \\
 & \Rightarrow \int_{r=0}^{r=R} \left(\frac{1}{r} \frac{\partial ru_r}{\partial r} \right) 2\pi r dr + \pi R^2 \frac{\partial \bar{u}_z}{\partial z} = 0 \\
 & \Rightarrow 2\pi R u_R + \pi R^2 \frac{\partial \bar{u}_z}{\partial z} = 0.
 \end{aligned}$$

Re-arrange leads to the **equation 2**,

$$u_r = -\frac{R}{2} \frac{\partial \bar{u}_z}{\partial z},$$

where the notation \bar{u}_z denotes the average z -velocity across cross-section.

Equation 3 Assume negligible convective acceleration and no viscous losses, the Navier-Stokes z -momentum equation can be simplified as,

$$\rho \left(\frac{\partial u_z}{\partial t} + u_r \frac{\partial u_z}{\partial r} + \frac{u_\theta}{r} \frac{\partial u_z}{\partial \theta} + u_z \frac{\partial u_z}{\partial z} \right) = -\frac{\partial p}{\partial z} + \mu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial u_z}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 u_z}{\partial \theta^2} + \frac{\partial^2 u_z}{\partial z^2} \right] + \rho f_z \Rightarrow \rho \frac{\partial \bar{u}_z}{\partial t} = -\frac{\partial p}{\partial z}.$$

Derivation of PVW First, let $u_r = \frac{\partial \Delta R}{\partial t}$, this equates **equation 1** and **equation 2** and leads to **equation 4**

$$\underbrace{u_r = -\frac{R}{2} \frac{\partial \bar{u}_z}{\partial z}}_{\text{equation 2}} = \underbrace{\frac{\partial \Delta R}{\partial t} = \frac{R^2}{Eh} \frac{\partial p}{\partial t}}_{\text{equation 1}} \Rightarrow \underbrace{\frac{\partial \bar{u}_z}{\partial z} = -\frac{2R}{Eh} \frac{\partial p}{\partial t}}_{\text{equation 4}}$$

Next, differentiate **equation 3** and **equation 4** w.r.t. t ,

$$\begin{aligned}
 \rho \frac{\partial \bar{u}_z}{\partial t} = -\frac{\partial p}{\partial z} & \xrightarrow[\text{w.r.t. } t]{\text{differentiate}} \rho \frac{\partial^2 \bar{u}_z}{\partial t \partial z} = -\frac{\partial^2 p}{\partial z^2}, \\
 \frac{\partial \bar{u}_z}{\partial z} = -\frac{2R}{Eh} \frac{\partial p}{\partial t} & \xrightarrow[\text{w.r.t. } t]{\text{differentiate}} \frac{\partial^2 \bar{u}_z}{\partial z \partial t} = -\frac{2R}{Eh} \frac{\partial^2 p}{\partial t^2},
 \end{aligned}$$

which allows us to equate the R.H.S. as

$$\frac{\partial^2 p}{\partial z^2} = \frac{2R\rho}{Eh} \frac{\partial^2 p}{\partial t^2} \Rightarrow \frac{\partial^2 p}{\partial t^2} = \underbrace{\frac{Eh}{2R\rho}}_{c^2} \frac{\partial^2 p}{\partial z^2},$$

which can be subsequently re-arranged as the wave equation. Denote the term $\frac{Eh}{2R\rho} = c^2$, for which the term c is the expression of the wave speed of pressure (a.k.a. pulse wave velocity, PVW). By definition, PVW increases with the stiffness of the vessels and decreases with the radius of the vessel.